

Figure 1: Problem 3.

$Q = 0.1 \text{ m}^3/\text{s}$. For that purpose, an ideal pump of power $\dot{W} = 12 \text{ kW}$ is used. The ambient pressure is P_a , the length of the pipe is $L = 10 \text{ m}$ and the elevation angle is $\alpha = 30^\circ$. Assume turbulent flow (check this assumption after having done all the calculations).

- a) Use the energy equation, neglecting kinetic energy at the exit, to show that the diameter of the pipe d can be written as

$$d = \left[\frac{8\rho Q^2 f L}{\pi^2 \left(\frac{\dot{W}}{Q} - \rho g L \sin \alpha \right)} \right]^{1/5}.$$

Also show that the diameter-based Reynolds number can be written as a function of the flow rate Q as $\text{Re}_d = 4\rho Q/\pi d\mu$.

- b) Use the two above expressions and the appropriate roughness to calculate the diameter of the pipe by iterating on the Moody chart.
4. A stainless steel pipe of internal diameter $D = 5 \text{ cm}$ carries water with an average velocity $V = 1 \text{ m/s}$. Compute the head loss and pressure drop per unit of pipe length.
5. An ideal turbine extracts 250 W of power in the configuration shown in the figure. The pipes are made of wrought iron. What is the flow rate Q in m^3/h ? The atmospheric pressure is P_a .

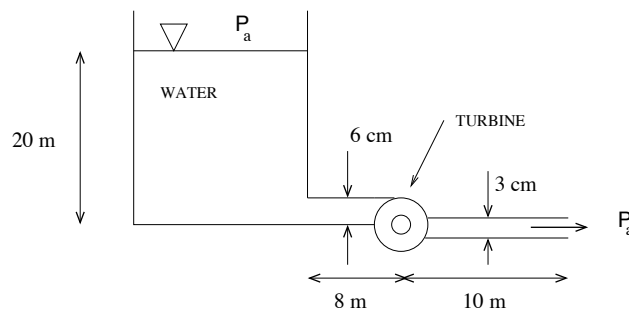


Figure 2: Problem 5.