

System Design: The Absentee in System Theory

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Dedicated to:

Osita Nwokah,

Friend, Scholar

With a Little Help From My Friends

Adhikari

Aldrich

Callafon

Grigoriadis

Helton

Kiichiro

Lu

Mingori

Murakami

Pinaud

Roman

Sato

Sultan

Williamson

Yamashita

Hubble Space Telescope (next servicing mission)


- New Solar Array
- Old Controller



Technology: What Paved the Way?

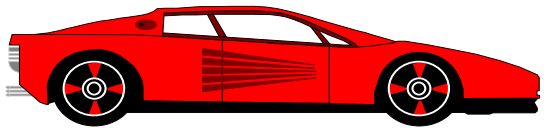
- 1st Half of 20th Century: **PHYSICS**
- 2nd Half of 20th Century: **ENGINEERING**
(Component Technology)
(Control is a component)
- Next? **SYSTEMS**
(Interdisciplinary Technology)

Outline

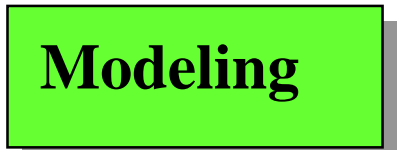
- **Designing Systems**
- **Designing Models for Systems**
- **Inspirations From**


ART
NATURE
- **Designing Controlled Structures**

Pin the Tail on the Performance Limiting Technology

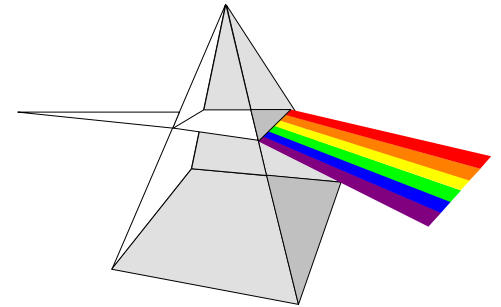
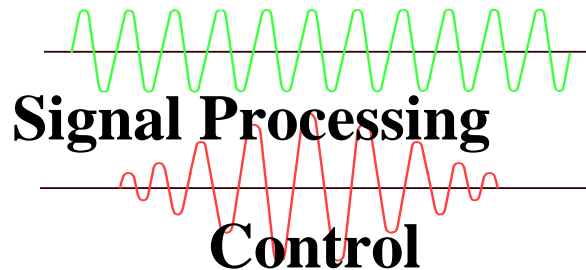
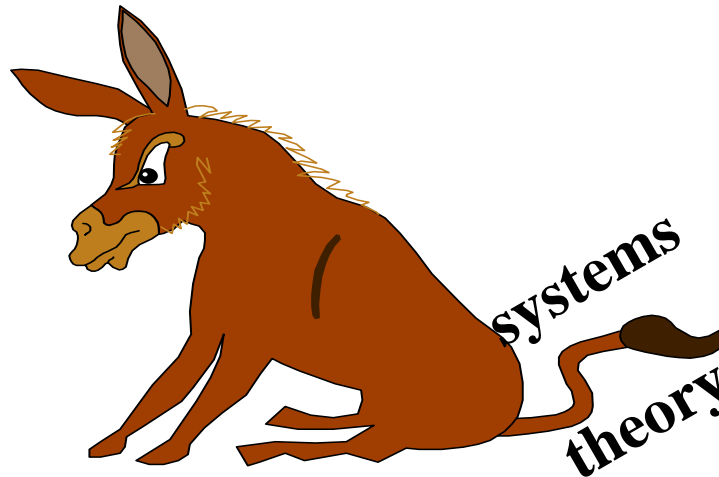
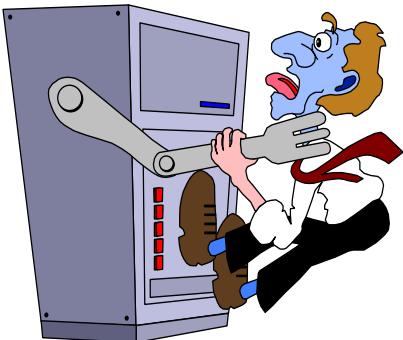


Manufacturing

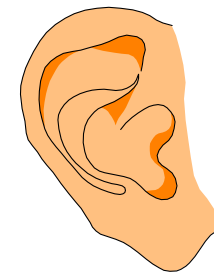


Modeling

Computing



Physics



Sensing

Systems Design Today

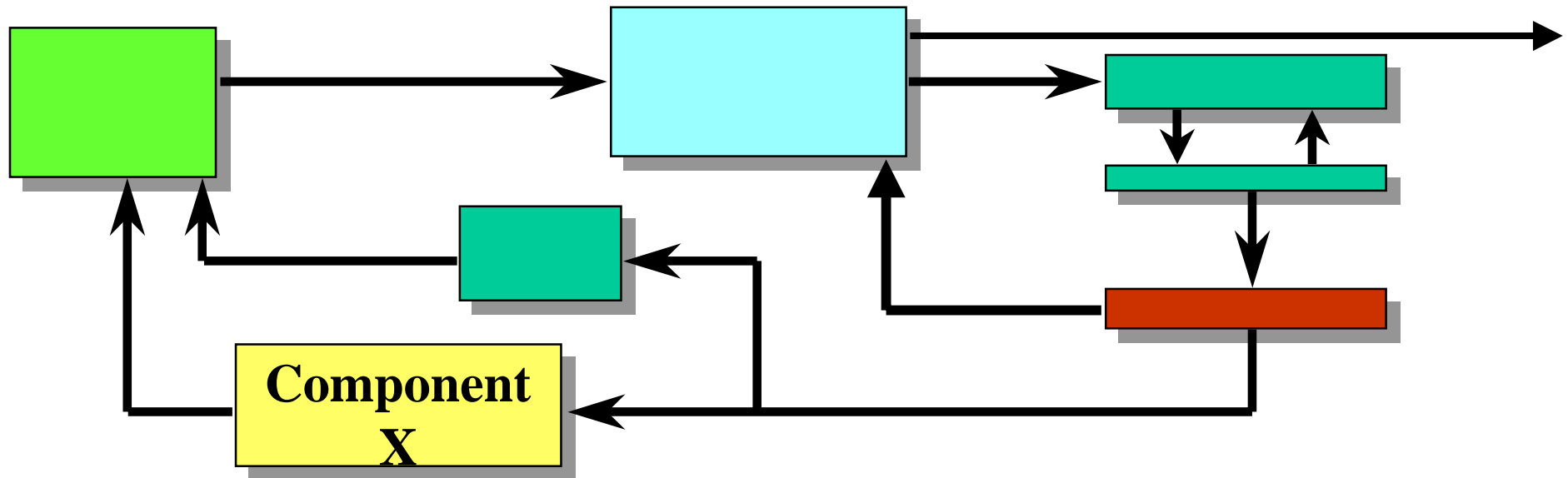
- **Universities Teach Component Technology**
 - materials, dynamics, signal processing, control
- **Leading to: uncoordinated multidisciplinary tasks**
 - **Manufacture** components, then
 - **Model** components (physics), then
 - **Connect, Measure, Actuate** components, then
 - **Control** the interconnected components
- **Problem:** dealing with sufficient rather than necessary
- **Systems** approach needed
- *Michael Faraday:* “**Begin with the whole,**
then construct the parts”

When is the Whole **LESS** Than the Sum of the Parts?

- The answer: **usually**
- Today, components are **overdesigned** to compensate for the lack of coordination in their design
- A Misconception: “**The best system is made from the best components**”
- Often, more gain in **integrating** two disciplines, than gained by **new technology** in either discipline
- Examples

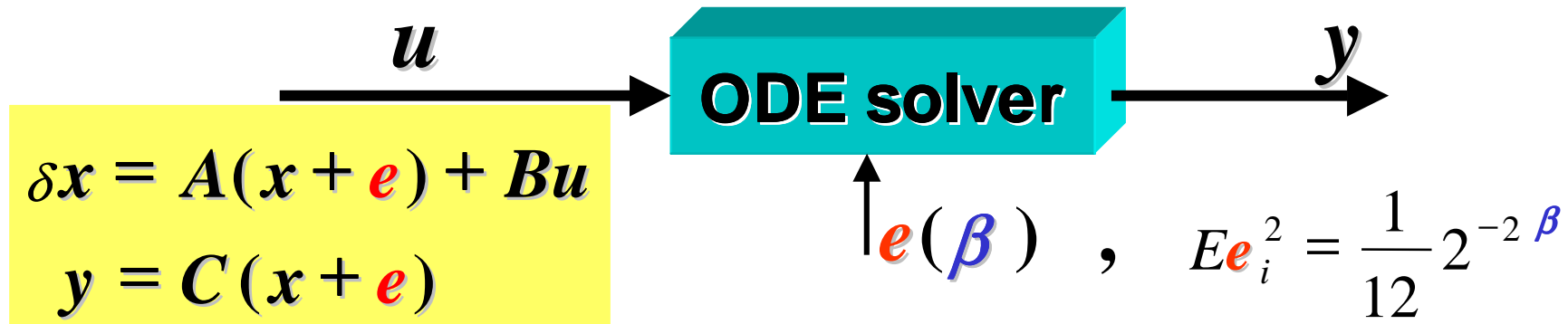
Given a System Requirement Where Should We Invest?

- How accurately to **Model** component X?
- How accurately to **Manufacture** component X?
- Is Component X even **Necessary**?
- How should the components be **Connected**?



- **Control** is a **Component** technology

Finite Precision Computing



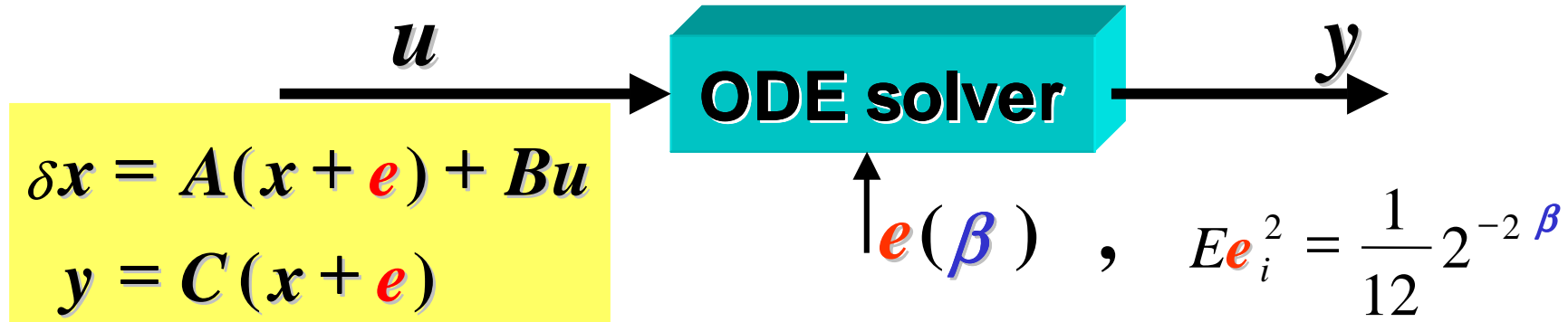
$x = T v$: No Clue about **Basis T** From Physics

$$y = G(z)u + G_e(z, T)e$$

- Useless to Model Better Than Error e (*unbounded over T*)

- How We.... **MODEL One Component** **Affects The Dynamics of Another**

Unified Signal Processing/Control



$$x = T v$$

No Clue about **Basis T** From Physics

$$y = G(z)u + G_e(z, T)e$$

$$\min_T \|G_e(T) e(\beta)\|$$

$$\exists (T^{-1} (Exx^T) T^{-T})_{ii} = 1$$

- [Mullis/Roberts 76]
- [Williamson 86]
- [Iiu/Grigoriadis/Skelton 88]
- [Gevers 92, Bamieh 94]

- **Component technology:** Design (A,B,C), then T
 - **Systems technology:** Design (A,B,C,T) jointly
- Control: Coupled ARE(β)

Hubble Space Telescope

- Limited by control
- Improvement @ no cost by unifying 2 disciplines

Using 24 bits: 100 times less (pointing variance)/(control energy)

Using 4 bits: 10,000 times less (“ ”)

change **existing** coefficients (no cost solution)

[IEEE TAC, Vol 37, No.9, 1992, JGCD, vol 18, No.2, 1995]

Optimal Mix of Plant/Control Design

[Grigoriadis, Zhu, Skelton, 1992]

$$\min_K E u^T u$$

$$E y y^T \leq Y$$

Guarantee
Performance Y

Convex, given Y

$$\underline{p}_i \leq p_i \leq \bar{p}_i$$

$$\begin{aligned} \dot{x} &= A(p)x + B(p)u + Dw, \\ y &= Cx \end{aligned}$$

$$[A \ B] = M + \sum p_i M_i$$

Optimal Mix of Plant/Control Design

[Grigoriadis, Zhu, Skelton, 1992]

$$\min_{\mathbf{p}, \mathbf{K}} \mathbf{E} \mathbf{u}^T \mathbf{u}$$

$$\mathbf{E} \mathbf{y} \mathbf{y}^T \leq \mathbf{Y}$$

Update
Plant \mathbf{p} ,
Control \mathbf{K}

Not Convex

$$\underline{p}_i \leq p_i \leq \bar{p}_i$$

$$\min_{\mathbf{K}} \mathbf{E} \mathbf{u}^T \mathbf{u}$$

$$\mathbf{E} \mathbf{y} \mathbf{y}^T \leq \mathbf{Y}$$

Guarantee
Performance \mathbf{Y}

Convex, given \mathbf{Y}

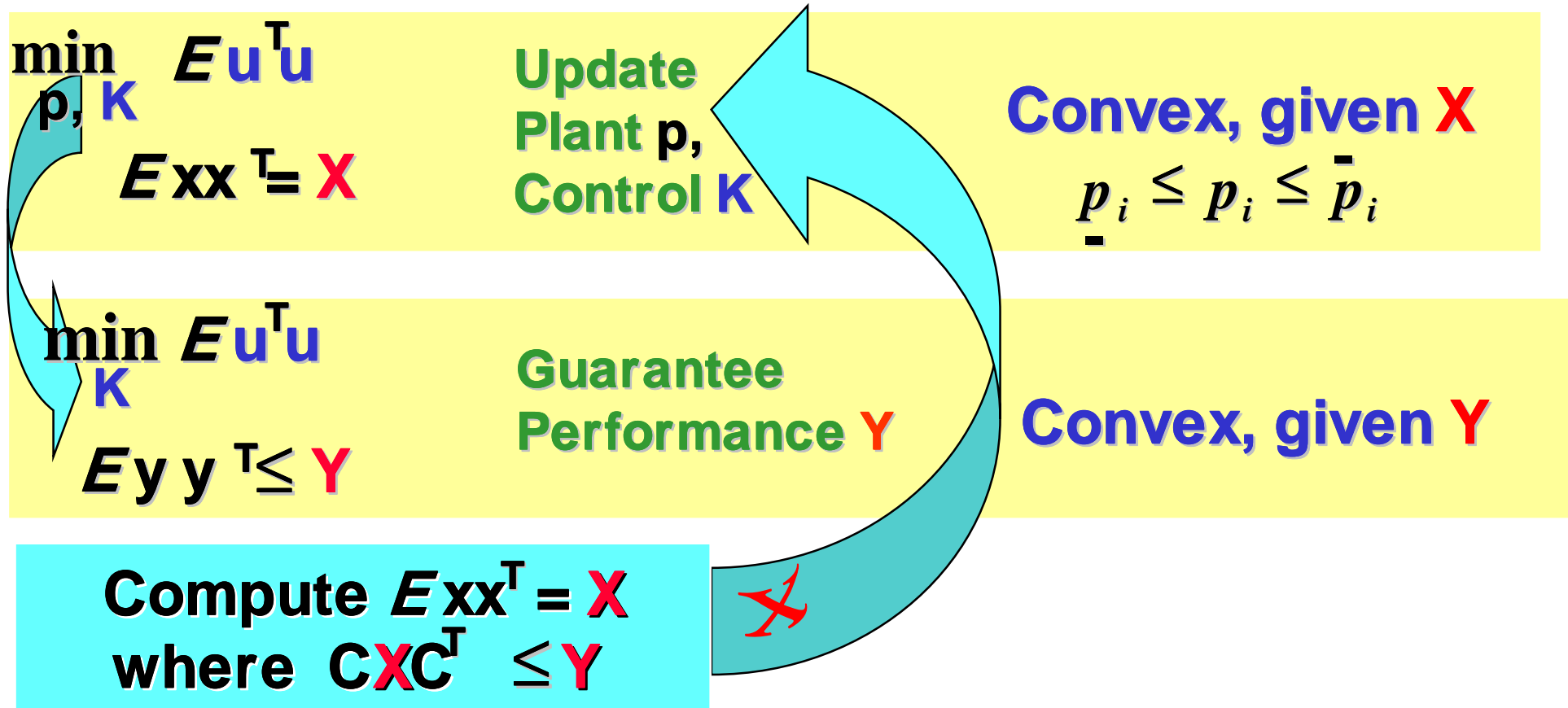
$$\dot{\mathbf{x}} = \mathbf{A}(\mathbf{p})\mathbf{x} + \mathbf{B}(\mathbf{p})\mathbf{u} + \mathbf{D}\mathbf{w},$$

$$\mathbf{y} = \mathbf{C}\mathbf{x}$$

$$[\mathbf{A} \ \mathbf{B}] = \mathbf{M} + \sum p_i \mathbf{M}_i$$

Optimal Mix of Plant/Control Design

[Grigoriadis, Zhu, Skelton, 1992]

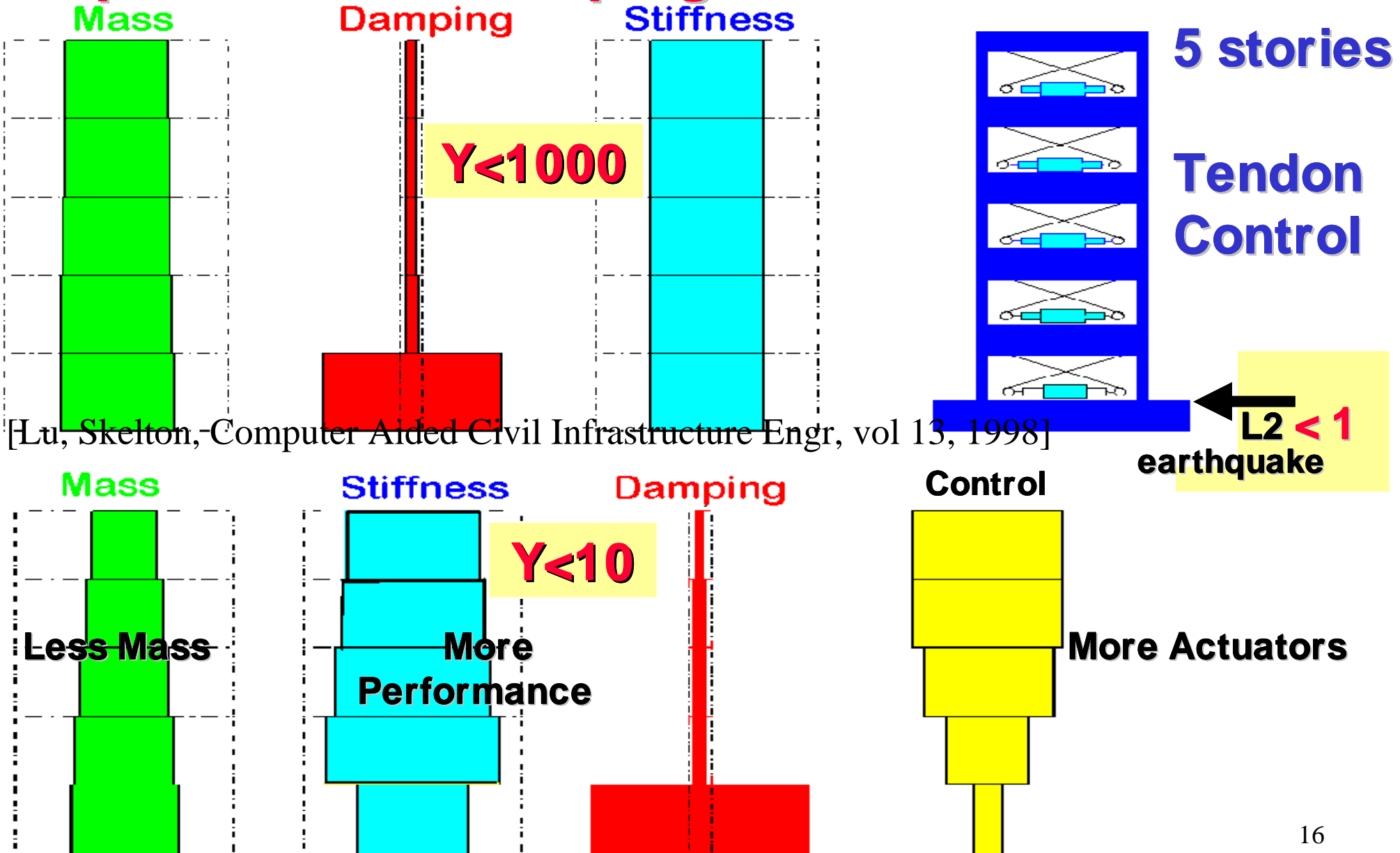


$$\dot{x} = A(p)x + B(p)u + Dw,$$

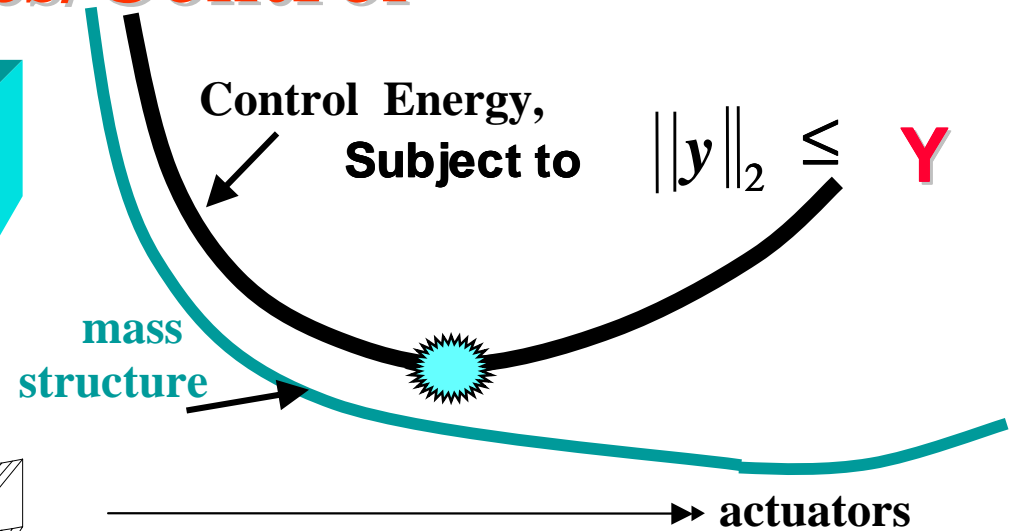
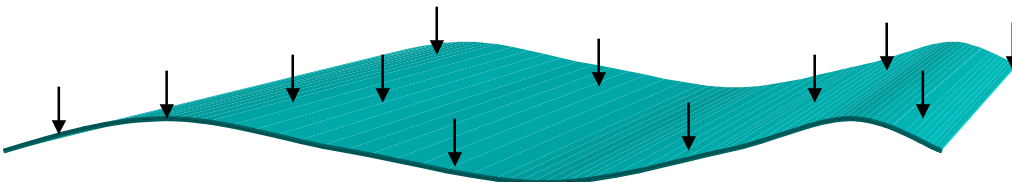
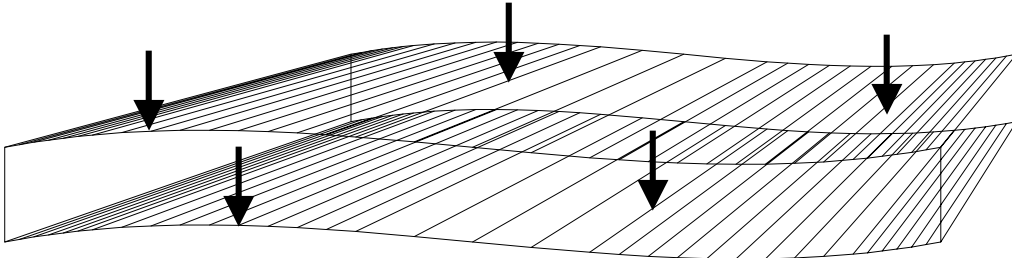
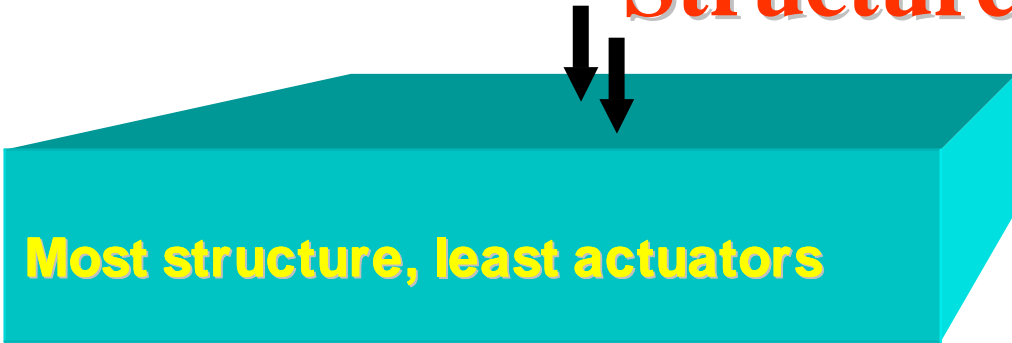
$$y = Cx$$

$$[A \ B] = M + \sum p_i M_i$$

Optimal Mass/Damping/Stiffness/Control



Optimal Mix of Physics/Information, Structures/Control



- Optimal **Number** Actuators
- Less energy, more robust
- More complex controller
- Avoiding integer program.....

The Economic Design Problem

$$W = \sigma Z$$

Finite Signal-to-Noise ratio

$$E w_j w_k = W \delta_{jk} \quad , \quad E z z^T = Z$$

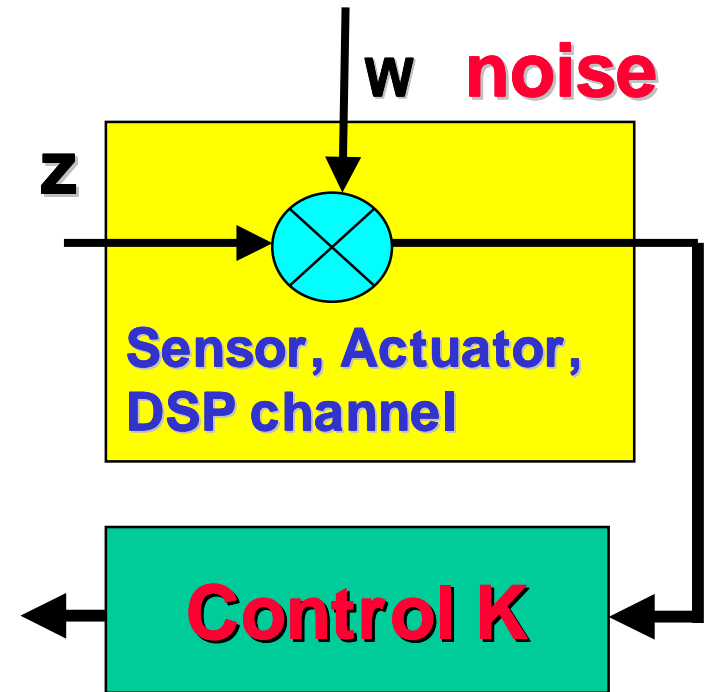
$$\$ = \sum_i p_i \sigma_i^{-1}$$

Min $\$$, subject to $E y y^T < Y$
 σ , K

Convergent algorithm, but no global

If $\sigma_1^{-1} \gg \sigma_2^{-1}$, then delete sensor 2, or off-the-shelf

[Lu, Skelton, Automatica, to appear]

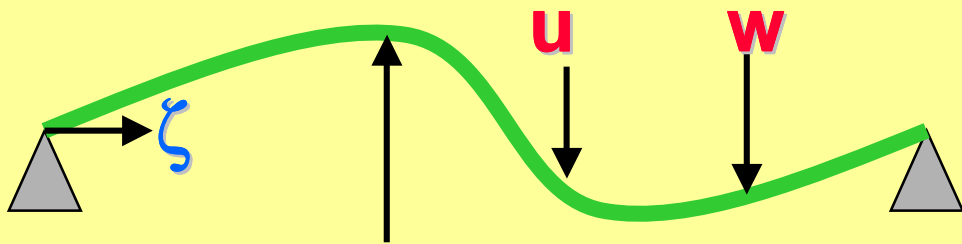
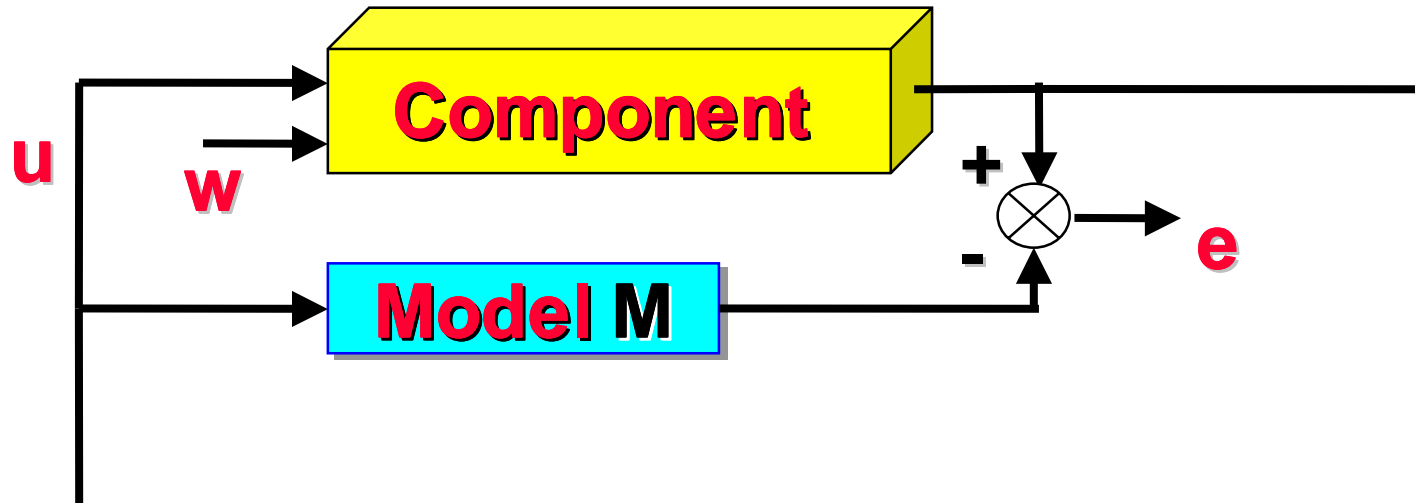


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- Designing Systems
- Designing Models for Systems
- Inspirations From **ART**
NATURE
- Designing Controlled Structures

Component Modeling

[Hu/Skelton, Computers and Structures, 1985]



- From **physics**,
Choose Φ for small e
- From **systems** criteria,
Choose Φ to depend on u, w

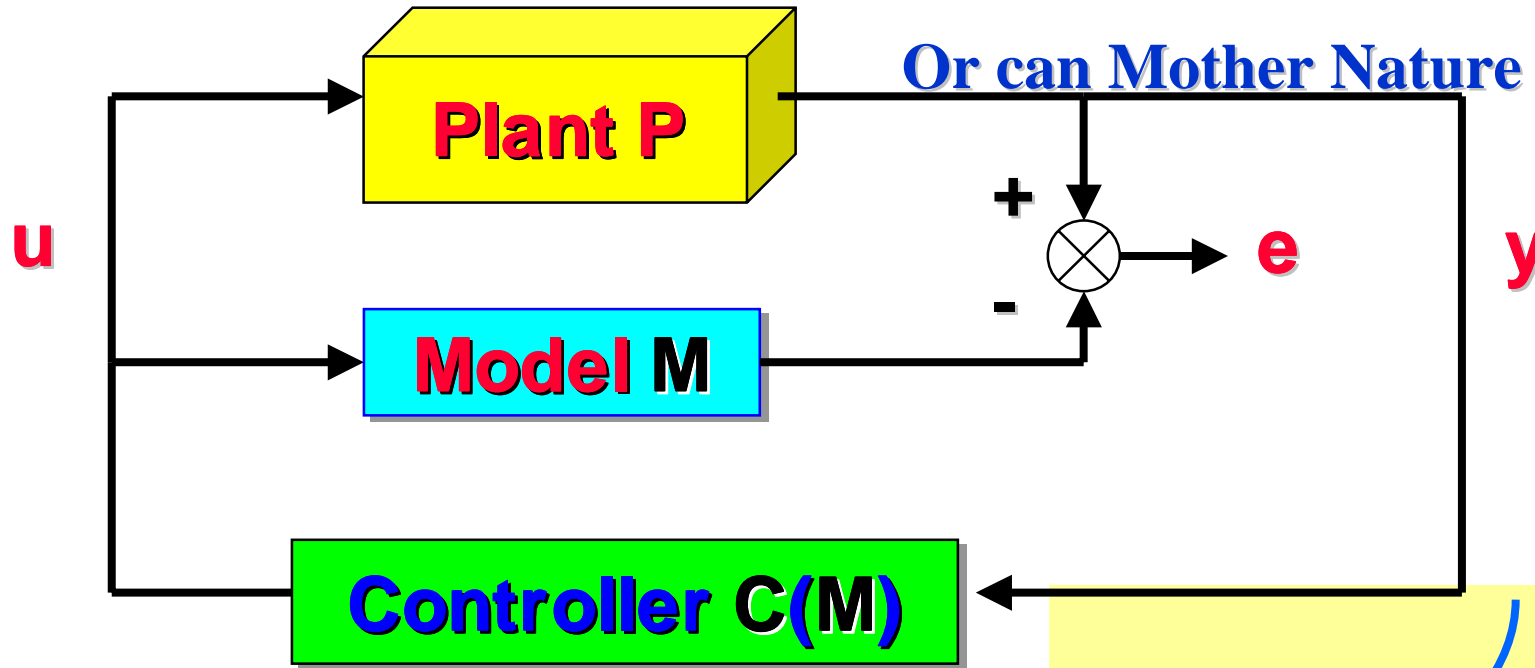
$$\mu(\zeta, t) = \Phi(\zeta) v(t)$$

recall $x = T v$

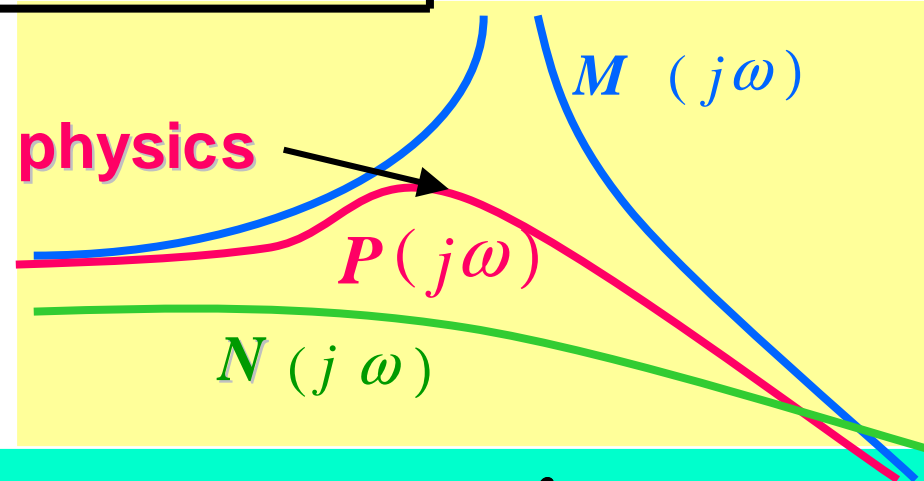
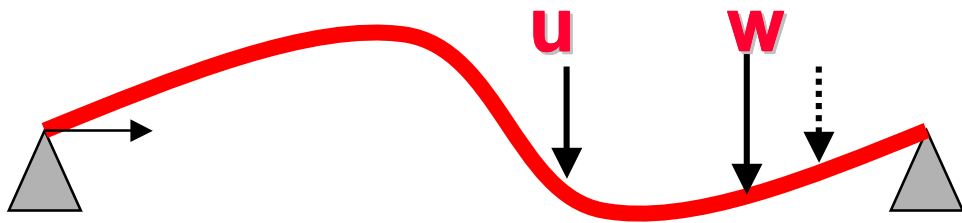
Choose T to depend on $w(b)$

Is System Modeling Just Physics, Physics?

Or can Mother Nature be fooled?



[Yousuff, 85]



$P \neq M + \Delta M$
 ↑
 Desired focus
 Robust Control Theory

$$C(P) = C(N) = C(M) \iff \int (\delta y^2 + u^2) dt$$

Control Models : How Much Info is Really Necessary?

$$\text{Minimize } \sum_{k=1}^N (y_k^T Q y_k + u_k^T R u_k)$$

$$\text{Subject to } x_{k+1} = A x_k + B u_k, \quad y_k = C x_k$$

[Shi, Skelton, DATA-BASED CONTROL, '94], [Furuta, '93], [Ikeda, '99]

Theorem

Optimal Controller Requires Only

$$CA^i B, \quad i = 0, 1, 2, \dots, N - 1$$

- **Only Errors in $CA^i B$ Affect Control Performance**
- **Any QMC from data yields the optimal control**
- **Why compute Markov Parameters, Use Data Directly**

Data Equivalent Models

[Skelton, Zhu, Qmarkov COVER, 1991]

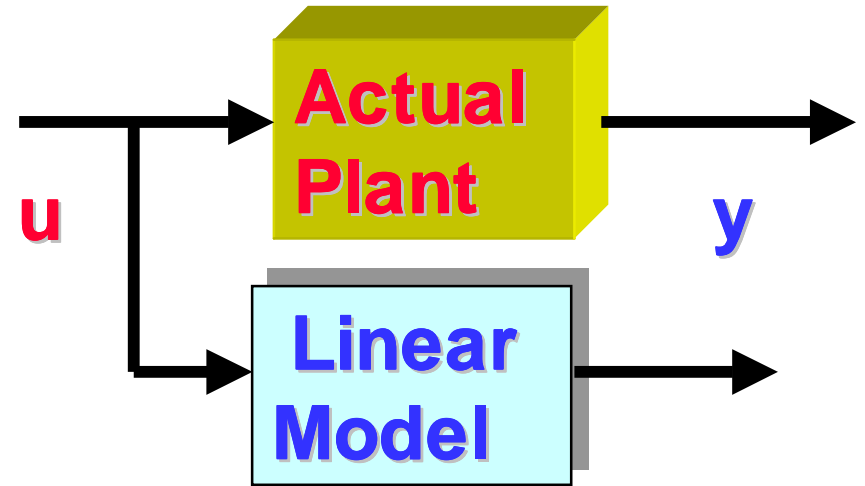
Does there exist any linear model to **Match** the **input/output** data?

IFF $R - HH^T \geq 0$

$$R_i = E y_{k+i} y_k^T, \quad H_i = E y_{k+i} u_k^T$$

$$R = \begin{bmatrix} R_0 & R_1^T & R_2^T & R_3^T \\ R_1 & R_0 & R_1^T & R_2^T \\ R_2 & R_1 & R_0 & R_1^T \\ R_3 & R_2 & R_1 & R_0 \end{bmatrix},$$

$$H = \begin{bmatrix} H_0 & 0 & 0 & 0 \\ H_1 & H_0 & 0 & 0 \\ H_2 & H_1 & H_0 & 0 \\ H_3 & H_2 & H_1 & H_0 \end{bmatrix}$$



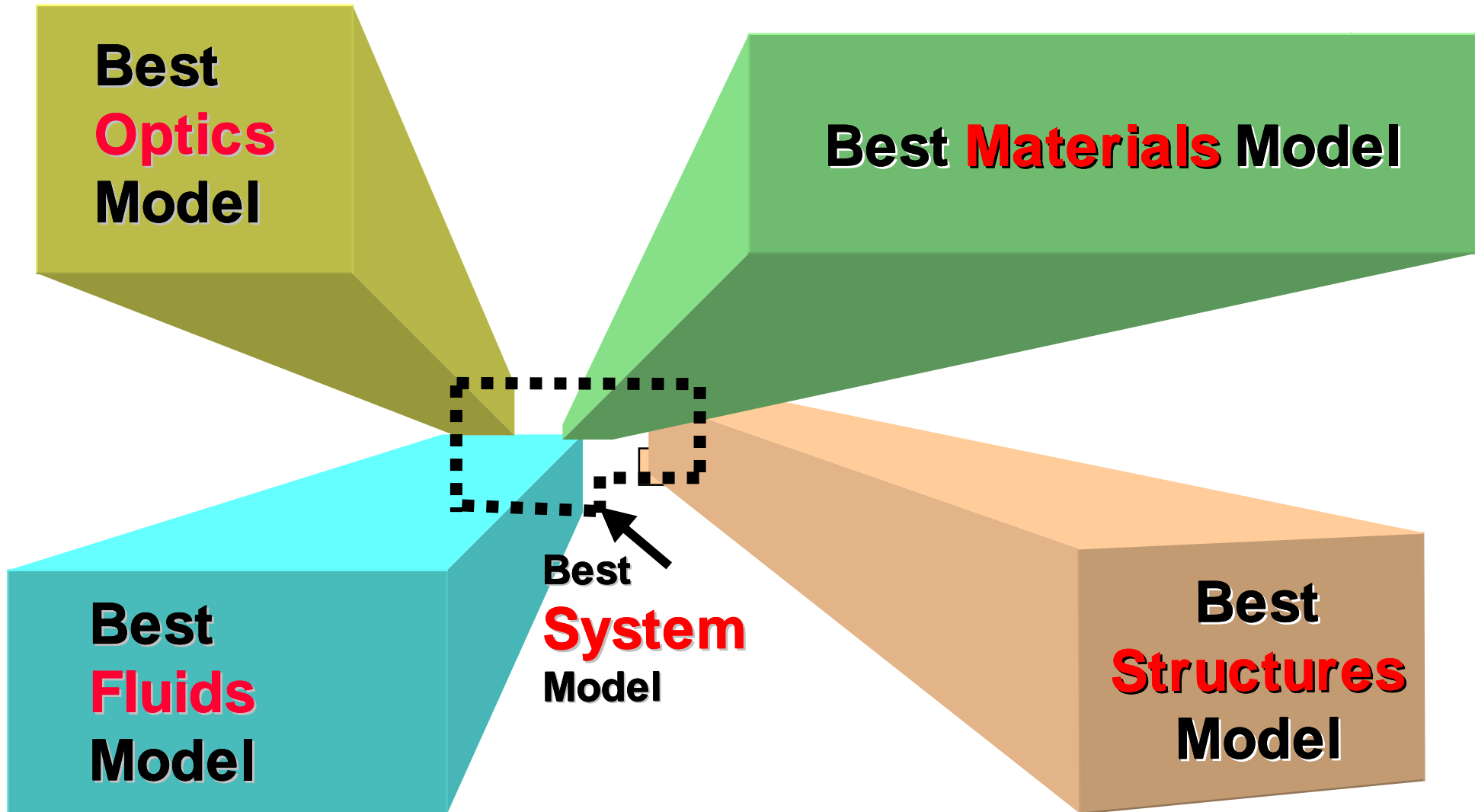
Many Models Equivalent to the One From Physics

Absentee: A System Modeling Theory

- The size of the error is **Not Continuous** from **Component (OL)** to **System (CL)**
 - Unbounded **OL** error, but zero **CL** error
 - Smaller **OL** errors \Rightarrow smaller **CL** errors. Hence....
- **Good Component Models** \Rightarrow **Good System Models**
 - There might exist a **Better** model for System Design than the **Actual** model from physics.
 - **Bad News:** Investments in Component Modeling may Not Help System Modeling
 - **Good News:** Good System Models can be Simpler Than Component Models (Yes, There Exists an Optimal Size)
- Should control design occur **Before, After, or During** plant modeling?

The Best System Model

- May have little to do with the best **Component Model**



Outline

- Designing **Systems**
- Designing **Models for Systems**
- Inspirations From **ART**
BIOLOGY
- Designing **Controlled Structures**

Integrating Material, Structure, Control

- The Theory of **Continua** is inadequate to account for the strength of materials.
- Challenge for Man-made Systems: **Architecture**
 - **Information Architecture**: selection of sensors, actuators, and feedback paths.
 - **Material Architecture**: selection of the material geometry
- Look to **Biological** and **Natural** systems, where
 - Mechanical, Chemical, and Electrical forces are involved in complex patterns of information flow, sensing, and feedback

Inspiration From Art

Kenneth Snelson

Needle Tower, 1968

Kroller Muller Museum

The Netherlands

“Tensegrity”
= Tension + Integrity

After 30 Years of

- **Forcing Continua**
- **Adding Actuators to**
Old Paradigms:
Beams, Plates, Shells

Eureka !!!!

- **No Joints**
- **No Load Reversals**
- **No Friction**
- **No Member Bending**
- **Easy to Change Equilibrium**



Mammalian Cell Cytoskeleton

Ingber, 98
Scientific American

Carbon Nanotubes, Fullerenes

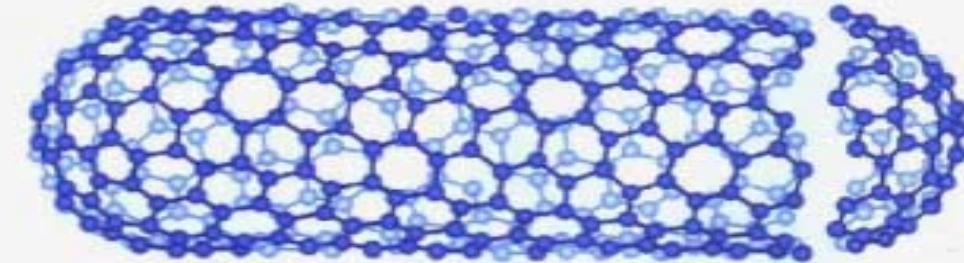
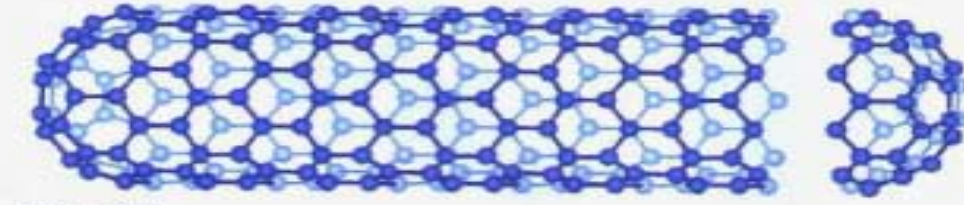
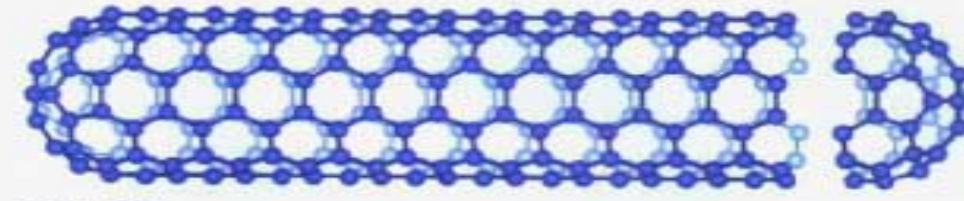
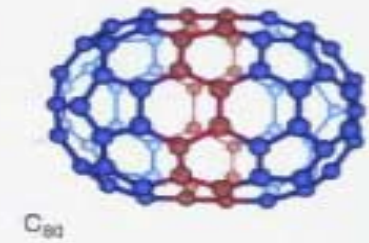
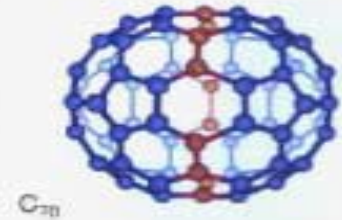
A scanning electron microscope (SEM) image showing a dense network of carbon nanotubes and fullerenes. The nanotubes appear as thin, interconnected fibers, while the fullerenes are represented by smaller, more rounded structures. The overall structure is a complex, interconnected mesh of carbon-based materials.

Strength From Geometry

A high-magnification scanning electron microscope (SEM) image of a single carbon nanotube. The nanotube is shown in a cross-section, revealing its hollow, cylindrical structure. The surface is composed of a regular, hexagonal lattice of carbon atoms, which gives it its unique strength and stability. The nanotube is shown in a slightly curved, cylindrical shape, highlighting its geometric structure.

[Yakobson, Smalley,
American Scientist, July 97]

Smalley, 1996 Nobel Prize



Buckyballs and Fullerenes

Geometry 

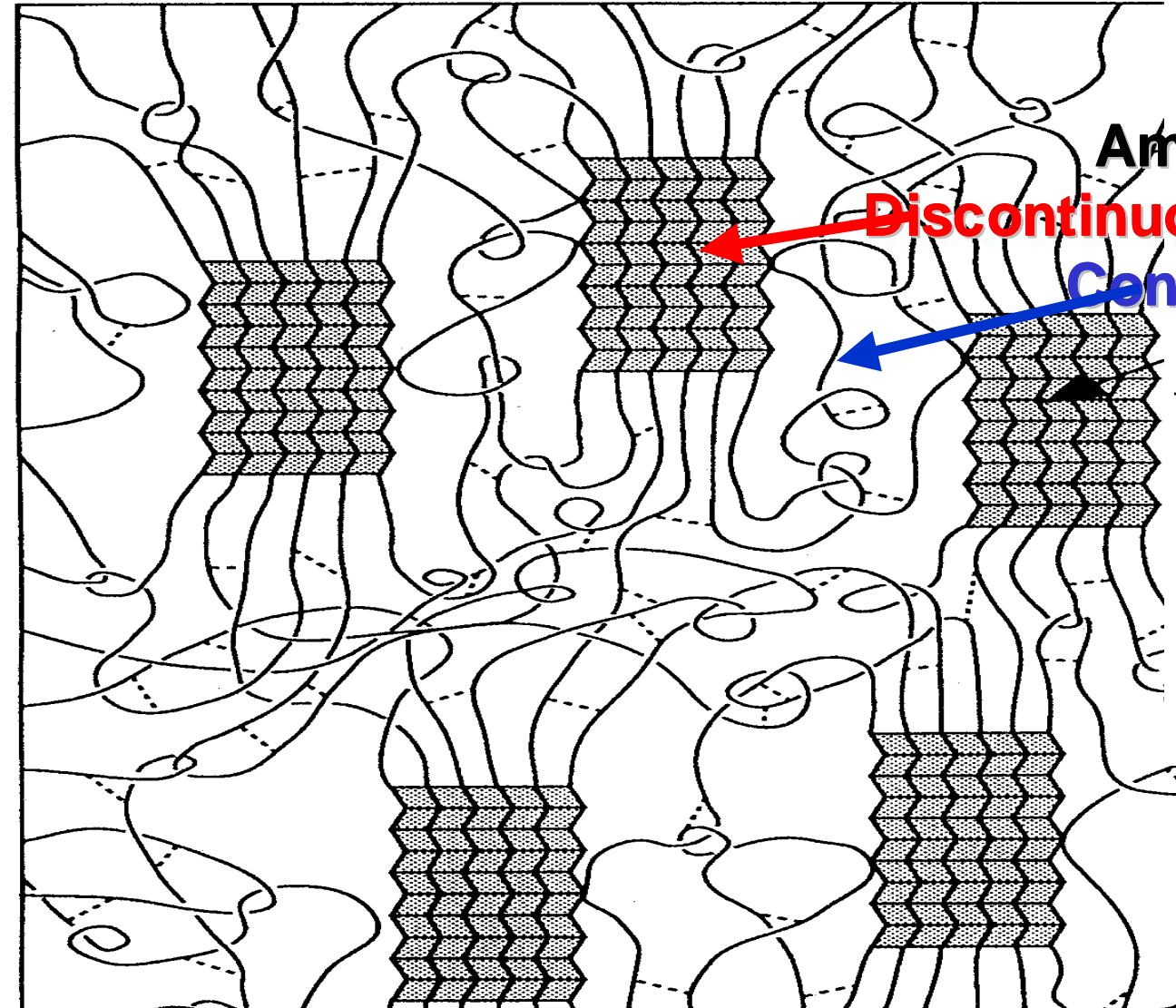
**mechanical properties,
electrical properties**

[Yakobson, Smalley,
American Scientist, July 97]



A Tensegrity Found in Nature,

Spider Fiber: Nature's Strongest



Amino Acids:

Discontinuous compressive

Continuous tension

What's in Common?

Fullerenes,

Spider fiber

Cell Cytoskeleton

Snelson's Artform

Tensegrity topology

Outline

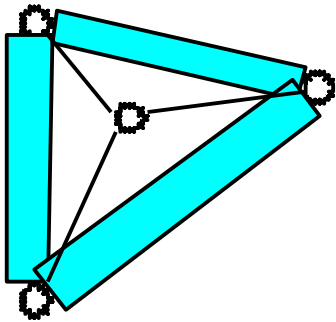
- Designing **Systems**
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Definition: Tensegrity Systems

2 N Points form a **Tensegrity Geometry** if the set of Points are stabilizable with pretensioned axially loaded members connecting the points, with no more than two compressive members attached to a Point

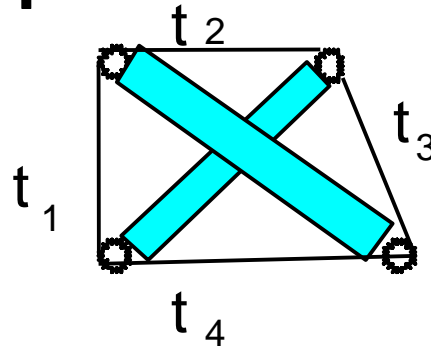
Class D: Discontinuous Compressive Members

Class C: Continuous Compressive Members



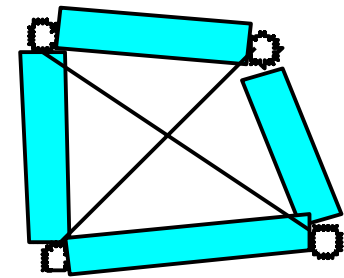
Tensegrity System

C3T3, Class C



Tensegrity System

C2T4, Class D

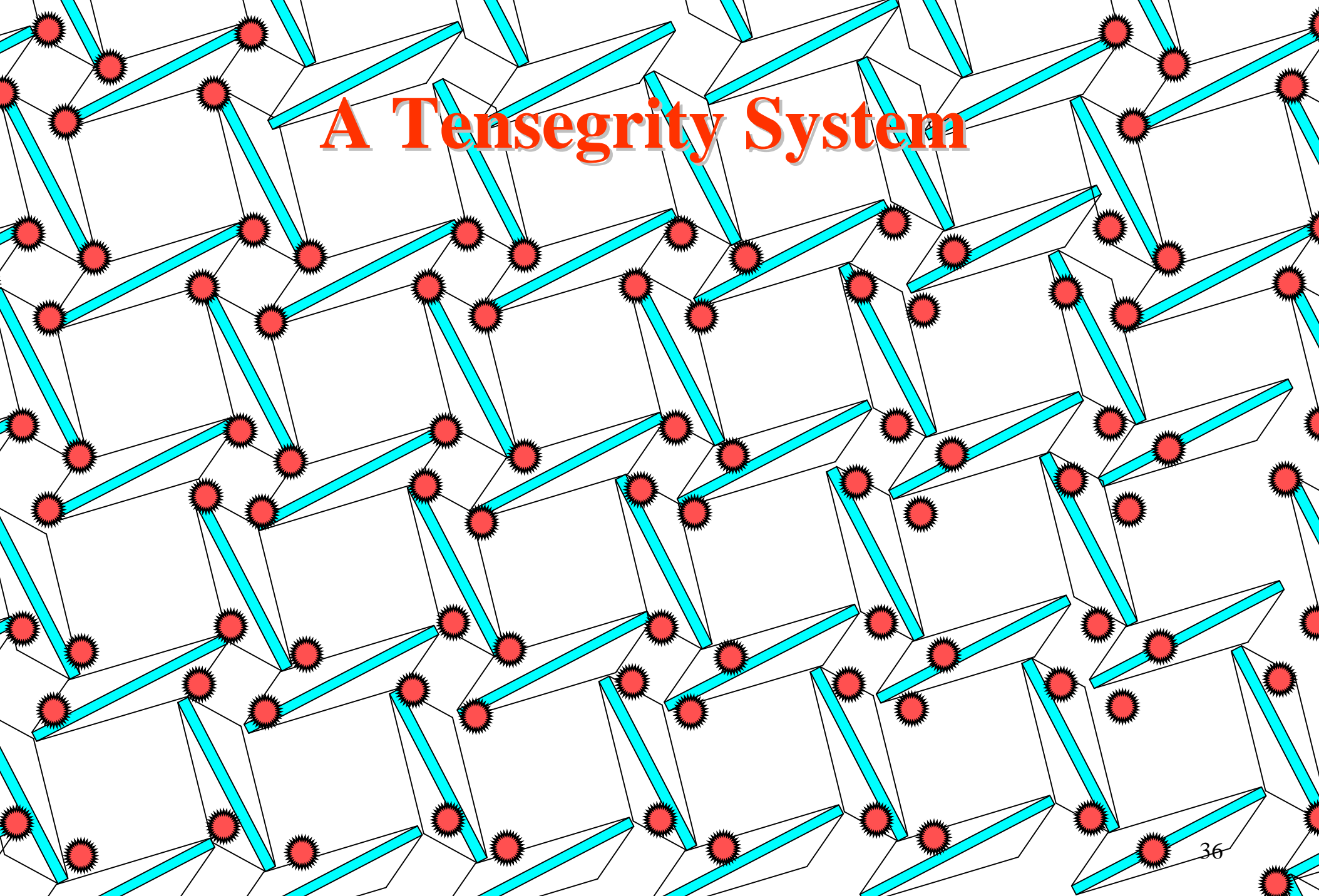


3D: No Tensegrity

A Tensegrity Geometry (To Form A Tube)

Points Not in a Plane

A Tensegrity System



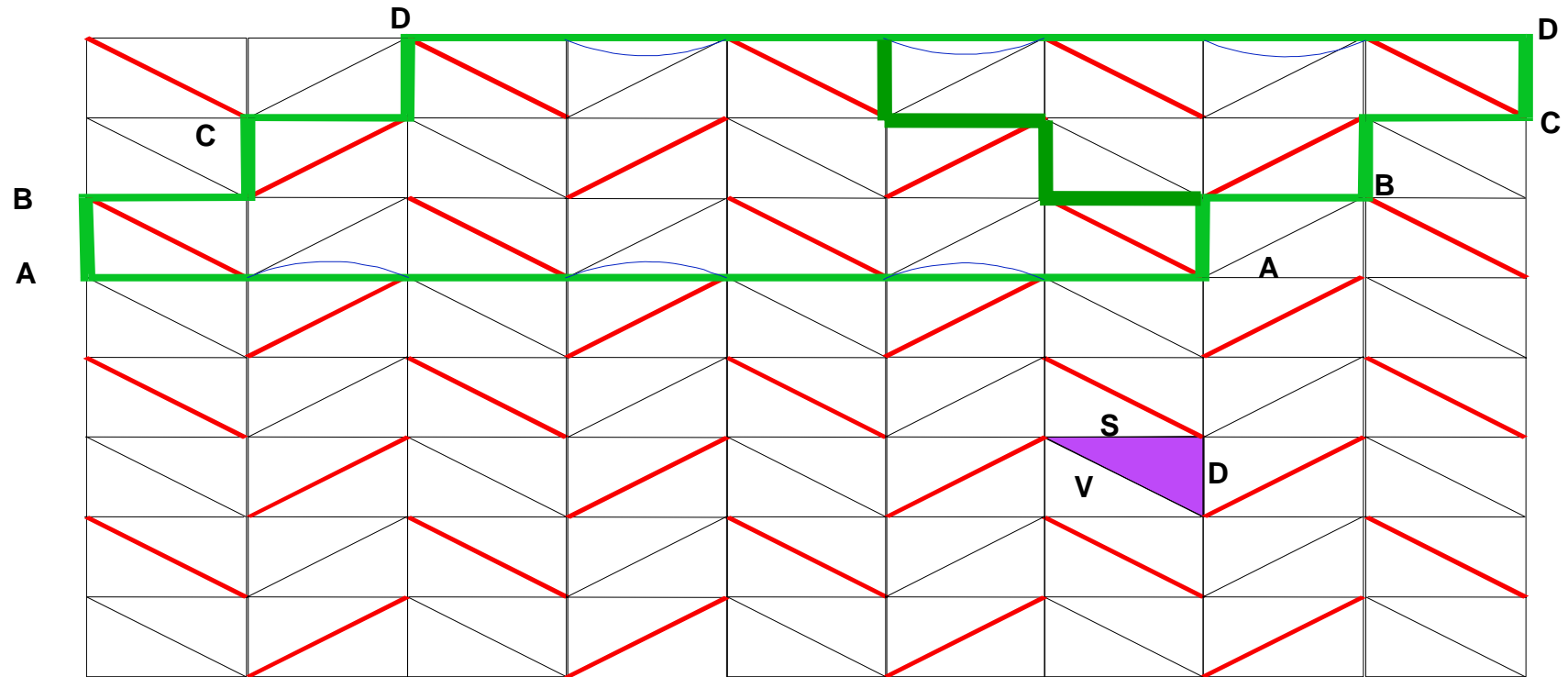
A Tensegrity System



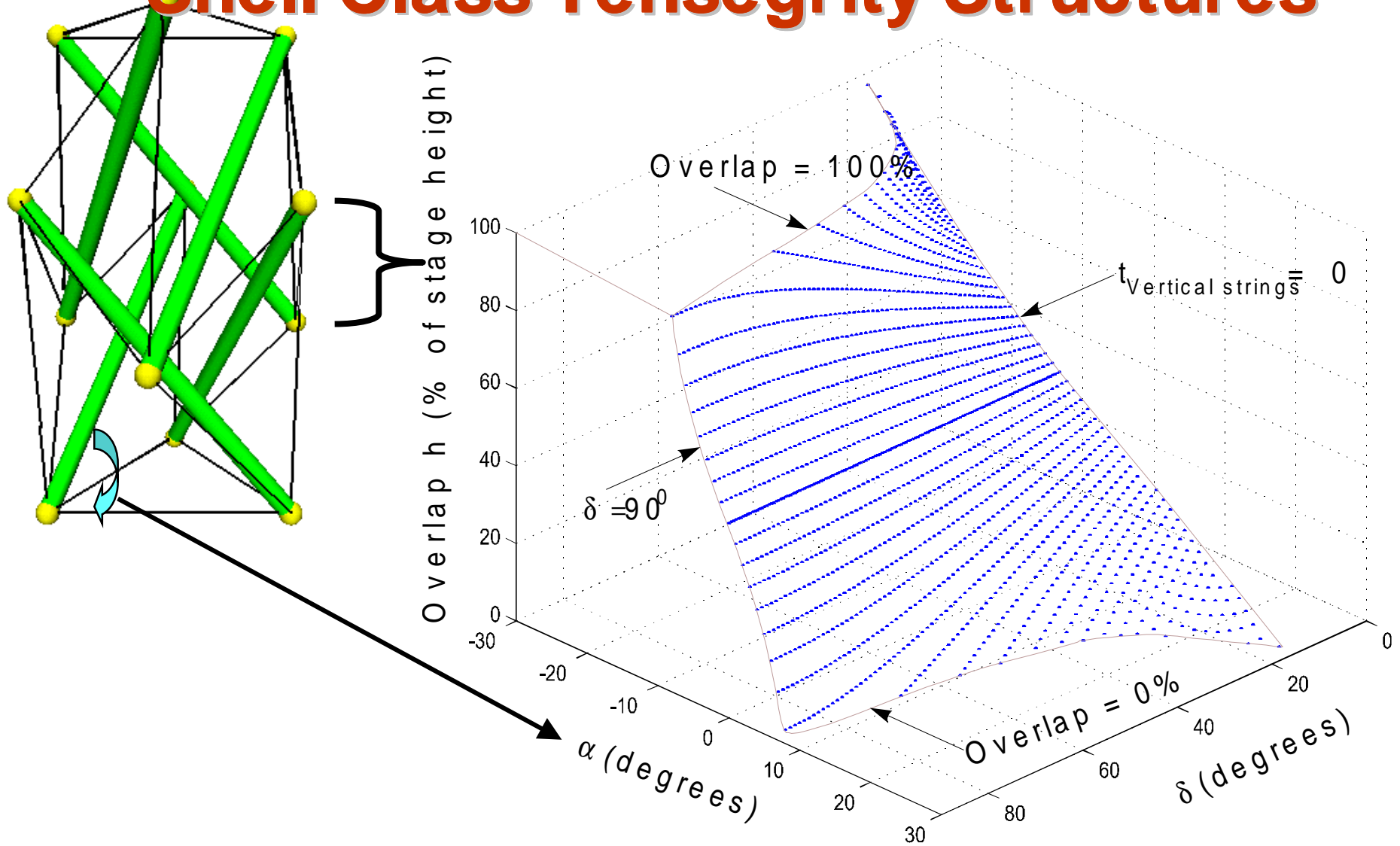
3 Local Parameters + Equilibrium Constraint

Symmetric Geometry: 2 Free Parameters

Shell Class of Tensegrity Systems



Shell Class Tensegrity Structures



Tensegrity Geometry

$$\begin{aligned}
 (h, \alpha, \delta) &= \text{Stable equilibrium} \\
 F(h, \alpha, \delta)t &= \mathbf{0}, \quad t > 0 \\
 |F^T F| &= \mathbf{0}
 \end{aligned}
 \left. \vphantom{\begin{aligned} (h, \alpha, \delta) \\ F(h, \alpha, \delta)t \\ |F^T F| \end{aligned}} \right\} \text{Tensegrity Geometry}$$

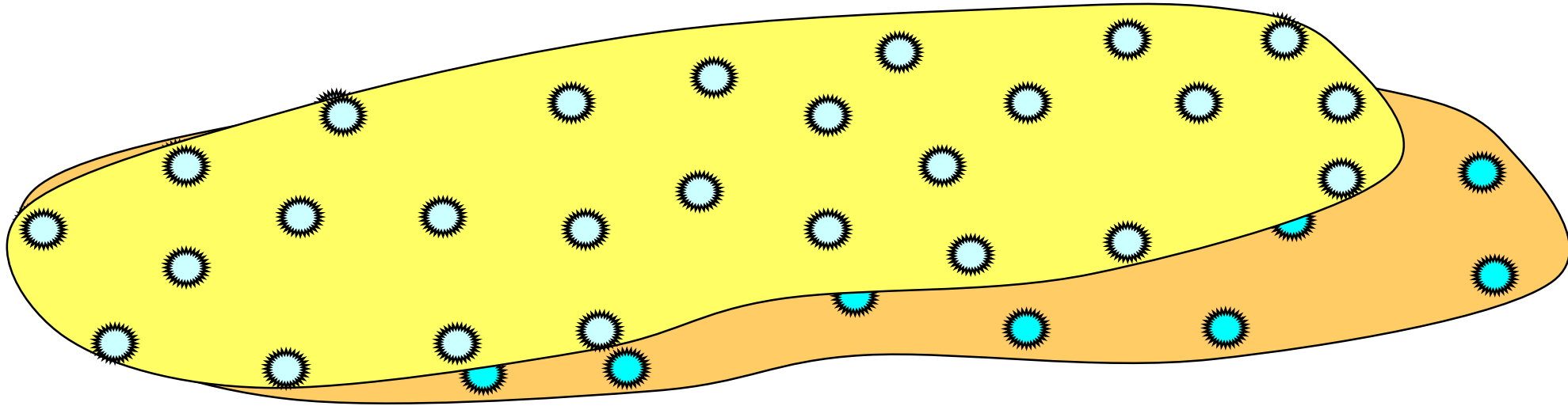
[Skelton, Helton, Adhikari, 1998],
[Sultan/Skelton, 1998]

$$h = \frac{1}{2 \tan \delta \cos \left(\alpha + \frac{\pi}{6} \right)} \left(-\frac{L_t}{\sqrt{3}} + L \sin \delta \cos \left(\alpha + \frac{\pi}{6} \right) + \sqrt{\frac{L_t^2}{3} - 3 L^2 \sin^2 \delta \cos^2 \left(\alpha + \frac{\pi}{6} \right)} \right)$$

- Pugh, 1976
- Pelligrino, Calladine, 1986
- Motro, 1986
- Furuya, 1992
- Coughlin, Stamenovic, 1997

- Skelton, 1993 - 1999
- Sultan, 1996, 1997, 1998, 1999
- Oppenheim, 1998
- Williamson, Skelton, 1999

Tensegrity Paradigm for Structural Control



- **Changing the Shape With Less Control Energy**
 - Construct a **Tensegrity Geometry** with a **specified** shape
 - Actuate the tendons (rest lengths) to avoid straining the structure, moving from one equilibrium to another

Shape Control with Tensegrity

Theorem [99]

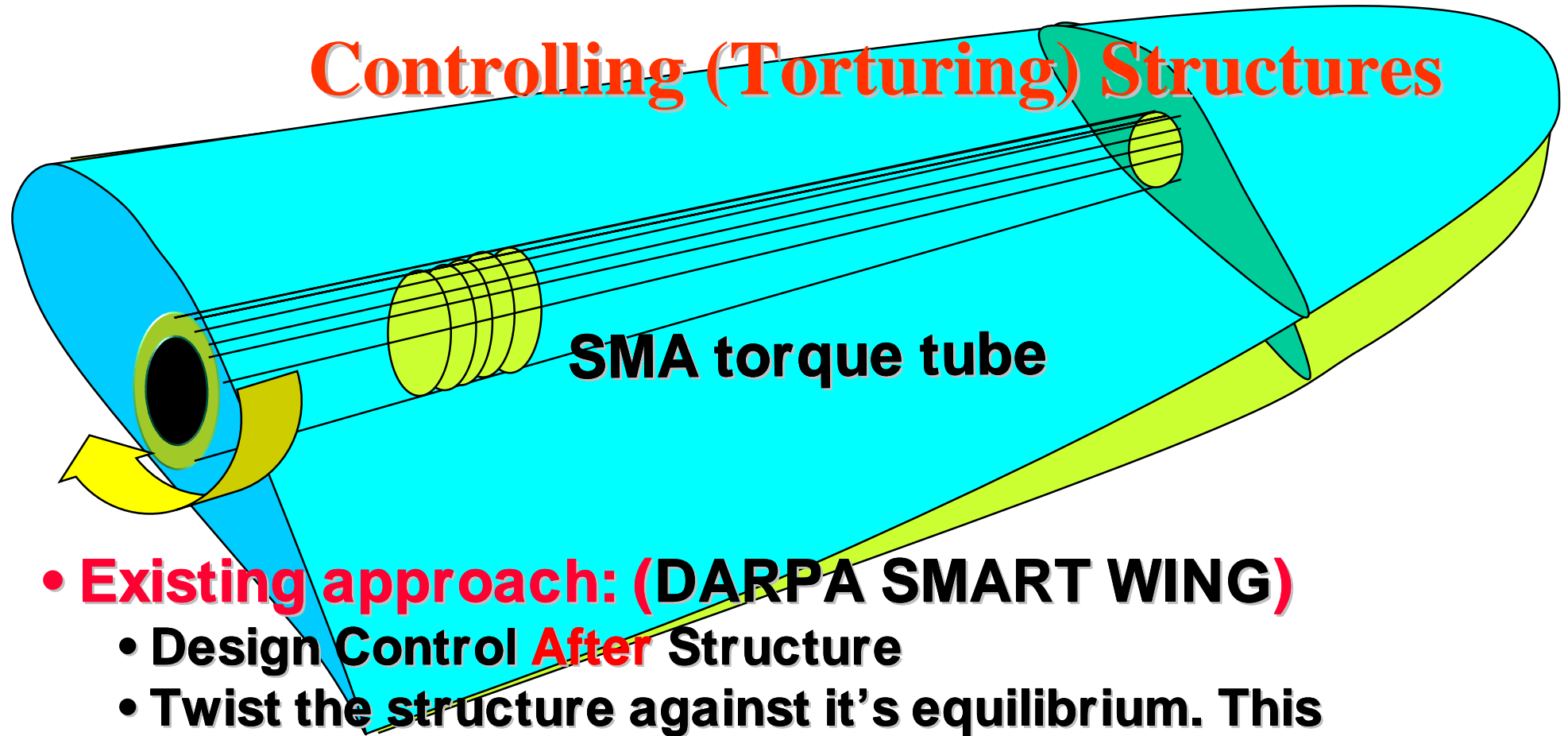
Let \mathbf{q}_1 , \mathbf{q}_2 be two tensegrity geometries, associated with the same nullspace of $\mathbf{F}(\mathbf{q})$. Then,

- There exists a continuum of tensegrity geometries between \mathbf{q}_1 and \mathbf{q}_2 .
- There exist tendon controls to change the geometry from \mathbf{q}_1 to \mathbf{q}_2 without changing potential energy.

$$\ddot{\mathbf{q}} + (K_r(\dot{\mathbf{q}}) + K_p(\mathbf{q}))\mathbf{q} = B(\mathbf{q})(\mathbf{u} + \mathbf{v}) + D\mathbf{w}$$

$$\dot{\mathbf{u}} = K^{-1}(\mathbf{R}\mathbf{q} - \mathbf{u}) \frac{(\mathbf{R}\mathbf{q} - \mathbf{u})^T KR \dot{\mathbf{q}}}{(\mathbf{R}\mathbf{q} - \mathbf{u})^T (\mathbf{R}\mathbf{q} - \mathbf{u})}, \quad \mathbf{R}\mathbf{q} > \mathbf{u}$$

Controlling (Torturing) Structures



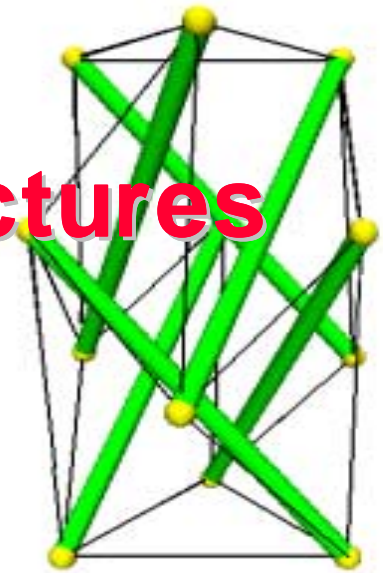
- **Existing approach: (DARPA SMART WING)**

- Design Control **After** Structure
- Twist the structure against it's equilibrium. This requires work (7 deg limit, 20 deg desired)

- **New Paradigm: Unify at more fundamental level**

- Change shape by changing the equilibrium

Minimal Mass Tensegrity Structures

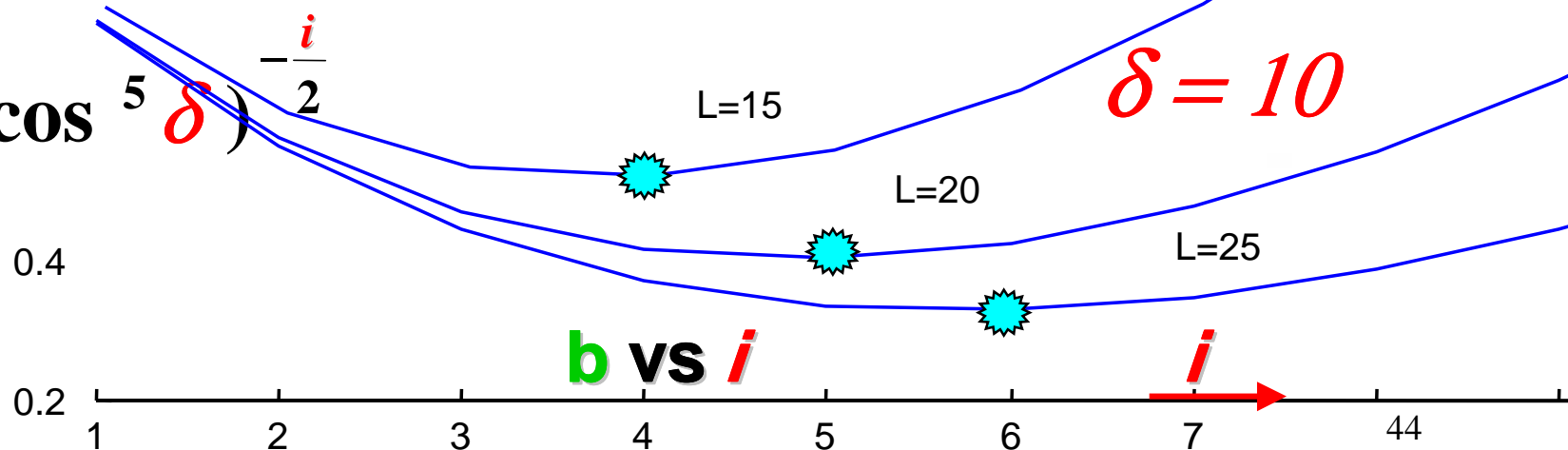


$$m_i = b m_0$$

$$b = \frac{\pi E_0 (\tan^2 \delta)}{\sigma} \left(\sum_{j=2}^i \frac{1}{(\cos^2 \delta)^{j-1}} \right) + m_B$$

$\sigma = \text{Tensile strength}$

$$m_B = (2 \cos^5 \delta)^{\frac{i}{2}}$$



Stiffness-to-Mass Ratio of C4T1^{*i*}

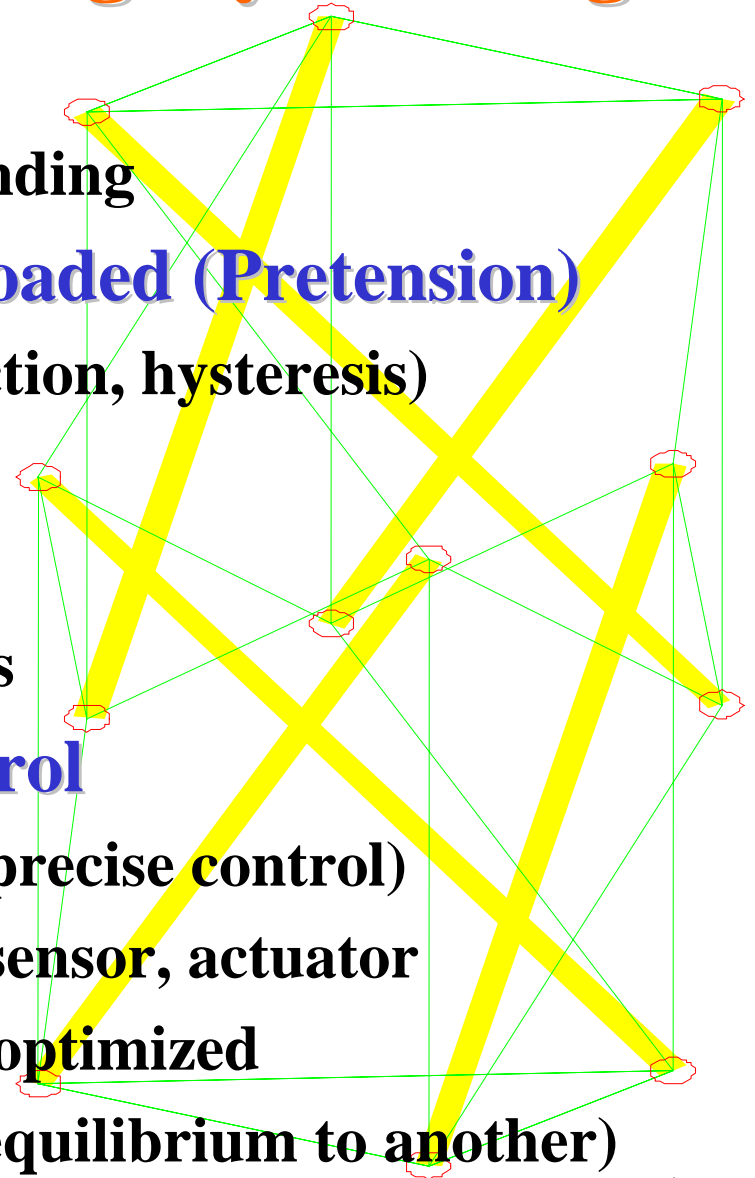
Theorem

The **compressive** stiffness of the C4T1^{*i*} Structure is equal to the **tensile** stiffness of the tendons of the shortest tendons

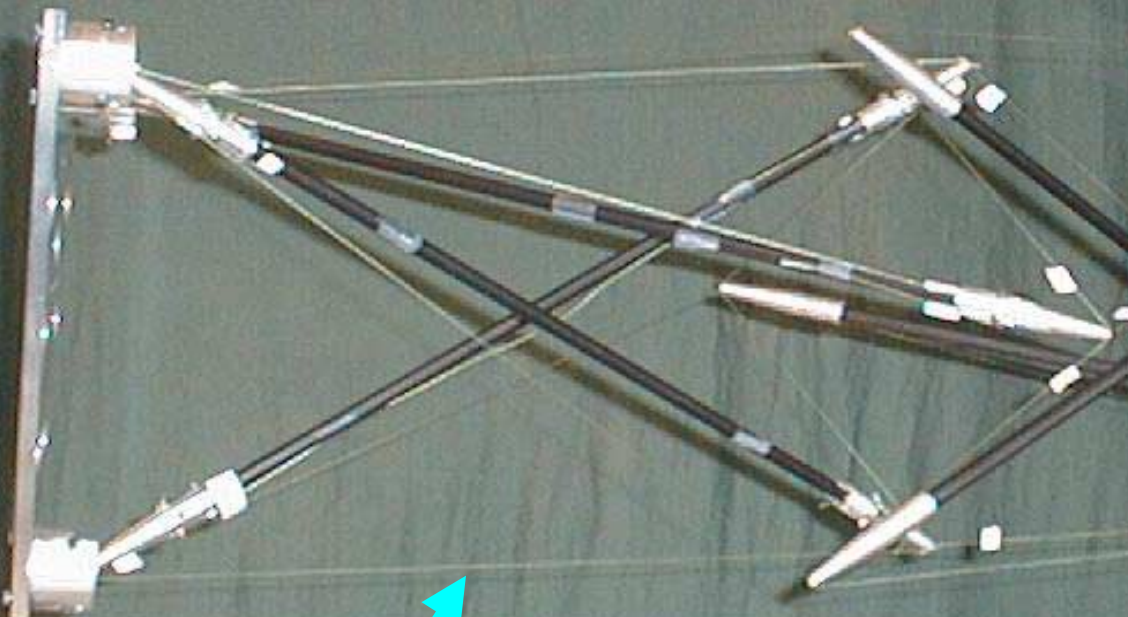
- Controlling the tension of the **shortest tendon** controls the compressive stiffness of the entire structure
- All compressive members carry the same load
- Infinite buckling strength for finite *i*
- minimal mass occurs at smaller *i* than infinite strength

Advantages of The Tensegrity Paradigm

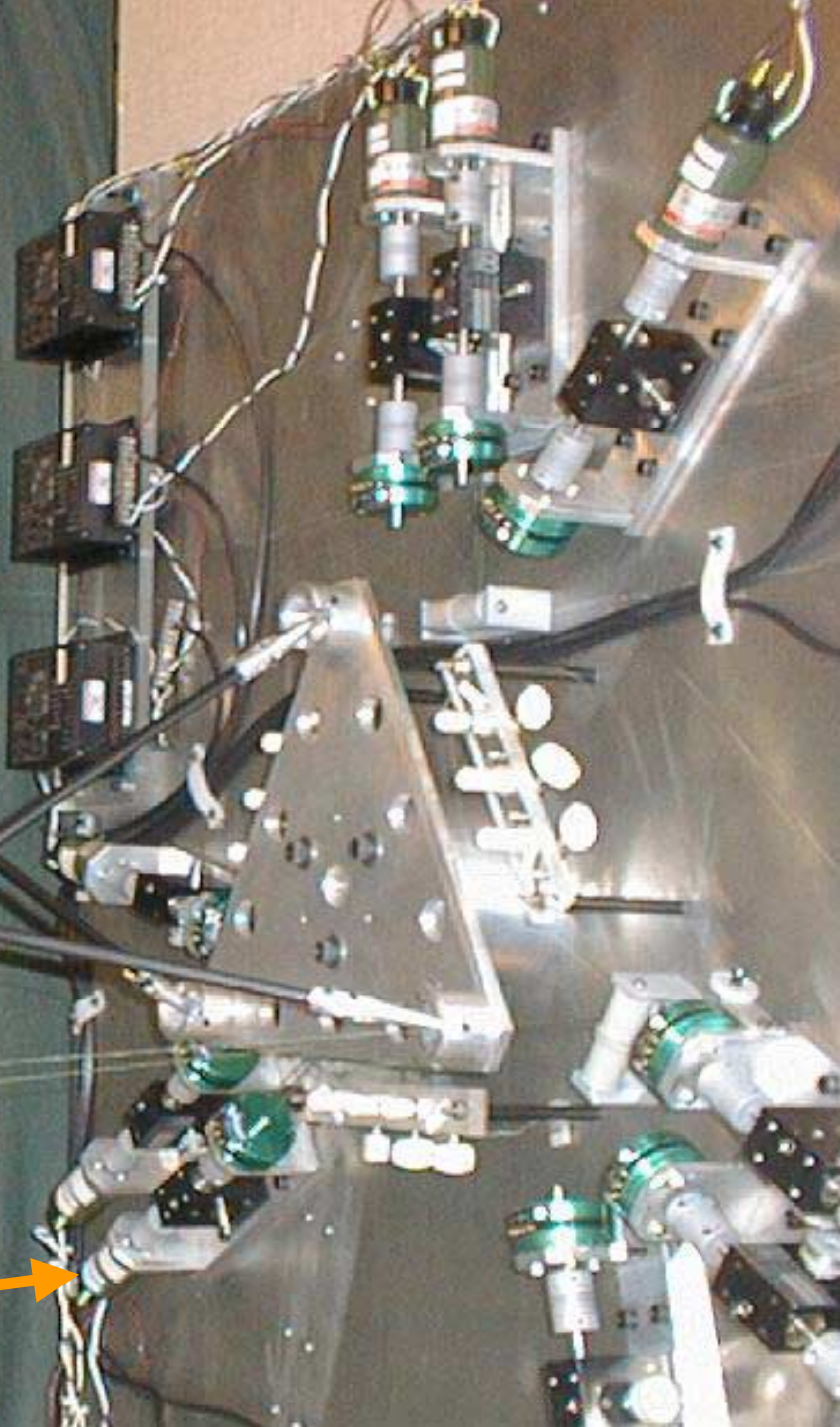
- **All Members Axially Loaded**
 - Global bending without member bending
- **All Members Uni-Directionally Loaded (Pretension)**
 - No reversal of load direction (no friction, hysteresis)
- **Structural Efficiency**
 - Strength to mass very high
 - Inspired by Art and Biological forms
- **Easy to Integrate Structure/Control**
 - More accurate models (hence more precise control)
 - A structural member also serves as sensor, actuator
 - Actuator/Sensor architecture easily optimized
 - Change shape with little work (one equilibrium to another)



Two-Stage Tensegrity: Tendon Control



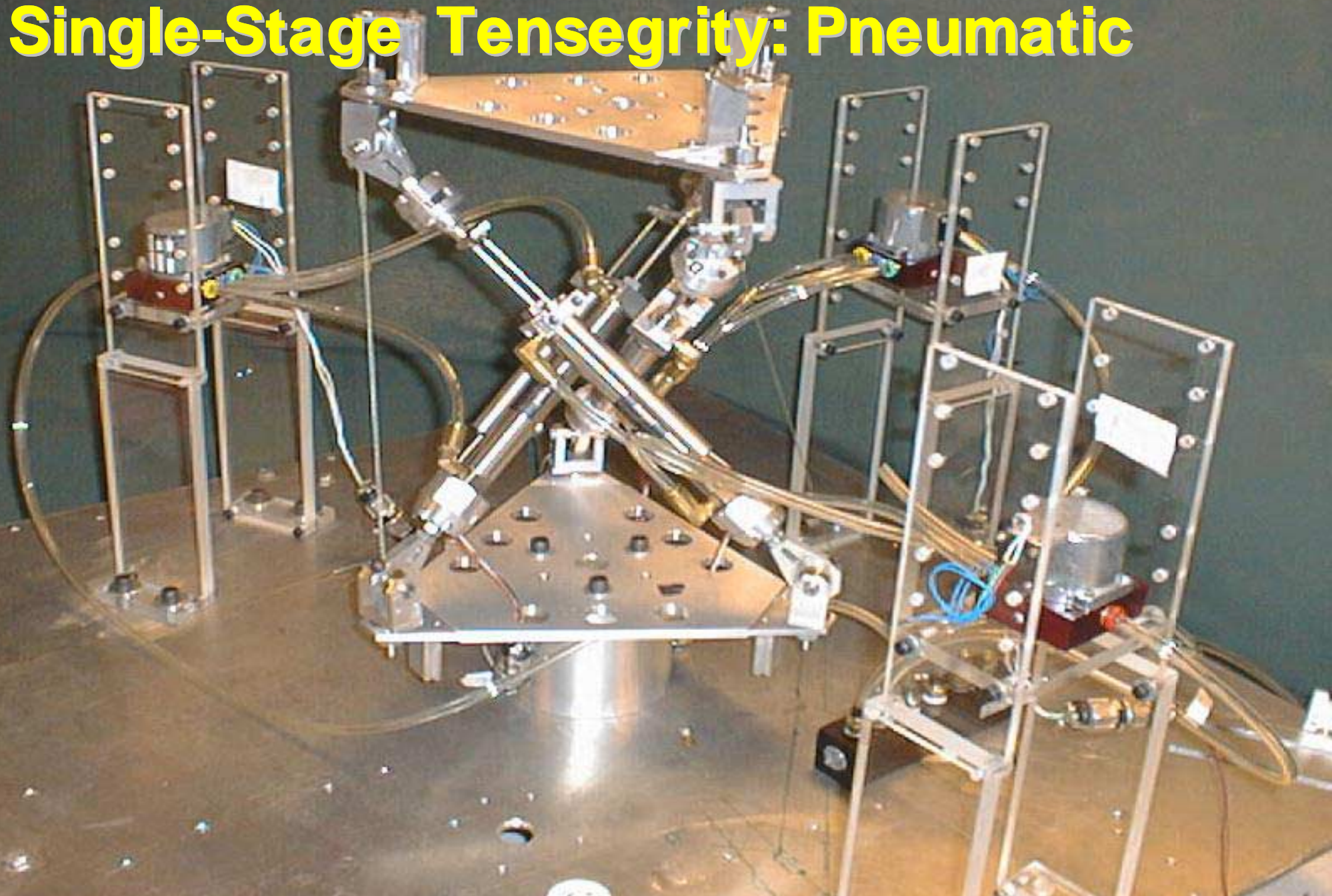
Controlled tendons
DC Motors



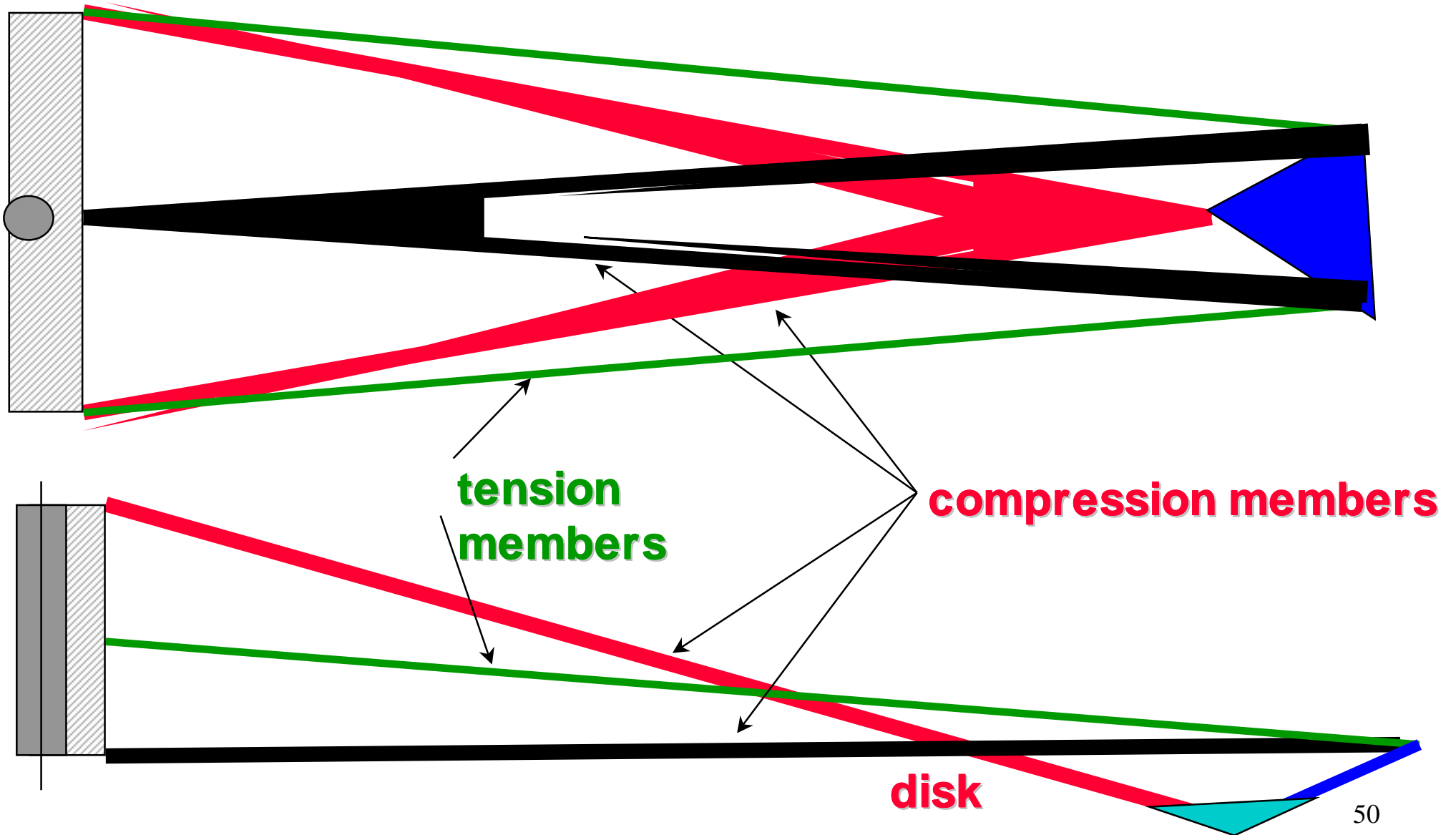
Deployable Tensegrity Heat Shield



Single-Stage Tensegrity: Pneumatic



Tensegrity Suspension for Disk Drives



Conclusions

To Universities and funding agencies:

- **Give the Soul of Control a Body: System Design**
- **Give Modeling a Purpose: Systems Modeling**
- **Function following Form**
 - **Snelson's Tensegrity Artform Inspires a New Paradigm to Integrate Mechanics, Structures, and Control**

Conclusions

- **After** Component Technologies mature, the next quantum leap in technology must come from a scientific method to do **Interdisciplinary System Design**
- Why wait til component maturity to invest in a scientific method for **Systems Design**
- The biggest challenge: **System Modeling**
- System Design (and modeling) requires **more** than communicating what each discipline **already knows**
- We can exploit **biological material architecture** to
- Suggest a **system design paradigm** for designing to specific materials, thermal, electrical, mechanical properties:

What Your Data Never Told You

- Data is not **Information**
- **Information** is not **Knowledge**
- **Knowledge** is not **Understanding**
- **Understanding** is not **Wisdom**

[Howard Garner, Harvard]